

TALAT Lecture 2301

Design of Members

Axial force and bending moment

Example 9.4 : Beam-column with cross weld

7 pages

Advanced Level

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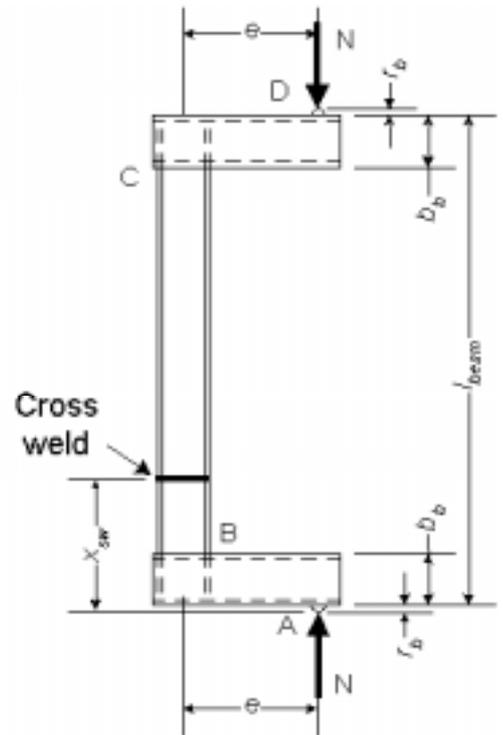
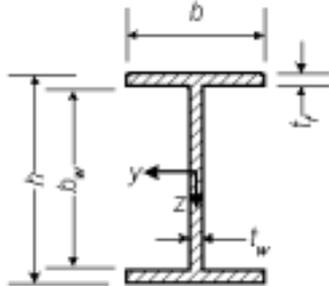
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Example 9.4. Beam-column with cross weld

The beam-column is subject to an eccentric axial force with same eccentricity at both ends. The beam is prevented from warping at the ends by rigid rectangular hollow beams. These beams are simply supported at the load point and D. The structure is a text beam.

There is a welded joint at a section x_{sw} from the support A.



Dimensions and material properties

Section height:	$h := 100 \cdot \text{mm}$
Flange depth:	$b := 49.9 \cdot \text{mm}$
Web thickness:	$t_w := 4.88 \cdot \text{mm}$
Flange thickness:	$t_f := 4.97 \cdot \text{mm}$
Overall length:	$l_{beam} := 1201 \cdot \text{mm}$
Cross beams:	$b_b := 140 \cdot \text{mm}$
Support	$r_b := 30 \cdot \text{mm}$
Eccentricity	$exc := 200 \cdot \text{mm}$
Location of cross weld	$x_{sw} := 170 \cdot \text{mm}$

Alloy: **EN AW-6082 T6** $heat_treated := 1$ (if heat-treated then 1 else 0)

Note: Values from tests!

$f_{0.2} := 311 \cdot \text{MPa}$ $f_{haz} := 223 \cdot \text{MPa}$

[1] (5.4), (5.5) $f_o := f_{0.2}$ $f_a := f_{haz}$ $E := 70000 \cdot \text{MPa}$ $G := 27000 \cdot \text{MPa}$
 [1] (5.6)

Partial safety factors: $\gamma_{M1} = 1.10$ $\gamma_{M2} = 1.25$

Inner radius: $r := 0 \cdot \text{mm}$

Web height: $b_w := h - 2 \cdot t_f - 2 \cdot r$ $b_w = 90 \cdot \text{mm}$

Moment and force

Axial force (compression)

$$N_{Ed} := 17 \cdot kN$$

Bending moment at the ends

$$M_{y,Ed} := N_{Ed} \cdot exc$$

$$M_{y,Ed} = 3.4 \cdot kNm$$

S.I. units:

$$kN \equiv 1000 \cdot newton$$

$$kNm \equiv kN \cdot m$$

$$MPa \equiv 1000000 \cdot Pa$$

Classification of the cross section

(Detail calculations omitted)

y-y-axis bending $class_y := 3$

z-z-axis bending $class_z := 3$

axial force $class_c := 3$

Cross weld

[1] 5.5 HAZ softening at a section $x_{sw} = 170 \cdot mm$ from the column end

[1] Table 5.2 $\rho_{ha\bar{z}} := 0.65$ In this example use value from tests $f_{ha\bar{z}} = 223 \cdot MPa$

which means $\rho_{ha\bar{z}} := \frac{f_{ha\bar{z}} \cdot \gamma_{M2}}{f_o \cdot \gamma_{M1}} \rho_{ha\bar{z}} = 0.815$ $f_u := \frac{f_{ha\bar{z}} \cdot \gamma_{M2}}{\rho_{ha\bar{z}} \cdot \gamma_{M1}} f_u = 311 \cdot MPa$

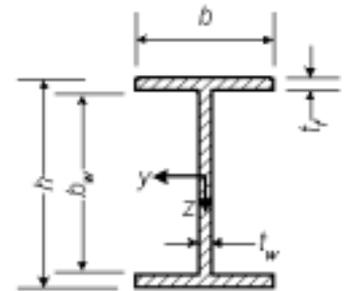
Design resistance, y-y-axis bending

[1] 5.6.1 Elastic modulus of the gross cross section W_{el} :

$$A_g := 2 \cdot b \cdot t_f + (h - 2 \cdot t_f) \cdot t_w \quad A_g = 935.499 \cdot mm^2$$

$$I_g := \frac{1}{12} \left[b \cdot h^3 - (b - t_w) \cdot (h - 2 \cdot t_f)^3 \right] \quad I_g = 1.418 \cdot 10^6 \cdot mm^4$$

$$W_{el} := \frac{I_g \cdot 2}{h} \quad W_{el} = 2.836 \cdot 10^4 \cdot mm^3$$



[1] Tab. 5.3

[1] (5.15) Shape factor α for class 3 cross-section, say

$$\alpha_y := 1$$

[1] (5.14) Design moment of resistance of the cross section $M_{c,Rd}$

$$M_{y,Rd} := \frac{f_o \cdot \alpha_y \cdot W_{el}}{\gamma_{M1}}$$

$$M_{y,Rd} = 8 \cdot kNm$$

Design resistance, z-z-axis bending

Cross section class

$$class_z = 3$$

Gross cross section:

$$I_z := 2 \cdot \frac{t_f b^3}{12}$$

$$I_z = 1.029 \cdot 10^5 \cdot mm^4$$

Section module:

$$W_z := \frac{I_z \cdot 2}{b}$$

$$W_z = 4.125 \cdot 10^3 \cdot mm^3$$

Shape factor:

$$\alpha_z := 1$$

Bending resistance:

$$M_{z,Rd} := \frac{f_o \cdot \alpha_z \cdot W_z}{\gamma_{M1}}$$

$$M_{z,Rd} = 1.166 \cdot kNm$$

Axial force resistance, y-y buckling

[1] 5.8.4 Cross section area of gross cross section A_{gr}

$$A_{gr} := b \cdot h - (b - t_w) \cdot (h - 2 \cdot t_f)$$

$$A_{gr} = 935.499 \cdot \text{mm}^2$$

Cross section area of class 3 cross section

$$A_{eff} := A_{gr}$$

Effective cross section factor $\eta := \frac{A_{eff}}{A_{gr}}$

$$\eta = 1$$

Second moment of area of gross cross section:

$$I_y := \frac{2}{12} \cdot b \cdot t_f^3 + 2 \cdot b \cdot t_f \left(\frac{h - t_f}{2} \right)^2 + \frac{1}{12} \cdot (h - 2 \cdot t_f)^3 \cdot t_w$$

$$I_y = 1.418 \cdot 10^6 \cdot \text{mm}^4$$

[1] Table 5.7 Effective buckling length

$$l_{yc} := l_{beam} + r_b$$

$$l_{yc} = 1.231 \text{ m}$$

Buckling load

$$N_{cr} := \frac{\pi^2 \cdot E \cdot I_y}{l_{yc}^2}$$

$$N_{cr} = 646.436 \cdot \text{kN}$$

[1] 5.8.4.1 Slenderness parameter

$$\lambda_{\bar{y}} := \sqrt{\frac{A_{gr} \cdot \eta \cdot f_o}{N_{cr}}}$$

$$\lambda_{\bar{y}} = 0.671$$

[1] Table 5.6 $\alpha := \text{if}(\text{heat_treated}=1, 0.2, 0.32)$

$$\alpha = 0.2$$

$\lambda_{\bar{\sigma}} := \text{if}(\text{heat_treated}=1, 0.1, 0)$

$$\lambda_{\bar{\sigma}} = 0.1$$

$$\phi := 0.5 \cdot \left[1 + \alpha \cdot (\lambda_{\bar{y}} - \lambda_{\bar{\sigma}}) + \lambda_{\bar{y}}^2 \right]$$

$$\phi = 0.782$$

$$\chi_{\bar{y}} := \frac{1}{\phi + \sqrt{\phi^2 - \lambda_{\bar{y}}^2}}$$

$$\chi_{\bar{y}} = 0.844$$

[1] Table 5.5 Symmetric profile

$$k_1 := 1$$

[1] Table 5.5 No longitudinal welds

$$k_2 := 1$$

Axial force resistance $N_{y,Rd} := \chi_{\bar{y}} \cdot k_1 \cdot k_2 \cdot \frac{f_o}{\gamma_{MI}} \cdot A_{gr}$

$$N_{y,Rd} = 223.4 \cdot \text{kN}$$

Axial force resistance, z-z axis buckling

	Effective buckling length	$l_{zc} := l_{beam} + r_b$	$l_{zc} = 1.231 \cdot m$
	Buckling load	$N_{cr} := \frac{\pi^2 \cdot E \cdot I_z}{l_{zc}^2}$	$N_{cr} = 46.9 \cdot kN$
[1] 5.8.4.1	Slenderness parameter	$\lambda_z := \sqrt{\frac{A_{gr} \cdot \eta \cdot f_o}{N_{cr}}}$	$\lambda_z = 2.49$
[1] 5.8.4.1		$\phi := 0.5 \cdot \left[1 + \alpha \cdot (\lambda_z - \lambda_o) + \lambda_z^2 \right]$	$\phi = 3.839$
		$\chi_z := \frac{1}{\phi + \sqrt{\phi^2 - \lambda_z^2}}$	$\chi_z = 0.148$
[1] 5.8.4.1	Axial load resistance	$N_{z,Rd} := \chi_z \cdot \eta \cdot k_1 \cdot k_2 \cdot \frac{f_o}{\gamma_{M1}} \cdot A_{gr}$	$N_{z,Rd} = 39.118 \cdot kN$
	and without column buckling	$N_{Rd} := \eta \cdot \frac{f_o}{\gamma_{M1}} \cdot A_{gr}$	$N_{Rd} = 264.5 \cdot kN$

Flexural buckling of beam-column

[1] 5.8.4.1	Cross weld from column end	$x_{sw} = 170 \cdot mm$	$l_{yc} = 1.231 \cdot m$	$\frac{x_{sw}}{l_{yc}} = 0.138$
[1] 5.9.4.5	HAZ reduction factors, see above	$\rho_{haz} = 0.815$		
[1] (5.51)		$\omega_{\bar{\rho}} := \rho_{haz} \cdot \frac{f_u}{\gamma_{M2}} \cdot \frac{\gamma_{M1}}{f_o}$	$\omega_{\bar{\rho}} := \text{if}(\omega_o > 1, 1, \omega_o)$	$\omega_{\bar{\rho}} = 0.717$
[1] (5.49)		$\omega_{\bar{x}} := \frac{\omega_o}{\chi_y + (1 - \chi_y) \cdot \sin\left(\frac{\pi \cdot x_{sw}}{l_{yc}}\right)}$		$\omega_{\bar{x}} = 0.788$

Exponents in interaction formulae

[1] (5.42c)	$\xi_o := \alpha_y^2$	$\xi_o := \text{if}(\xi_o < 1, 1, \xi_o)$	$\xi_o = 1$
[1] 5.9.4.2	$\xi_{yc} := \xi_o \cdot \chi_y$	$\xi_{yc} := \text{if}(\xi_{yc} < 0.8, 0.8, \xi_{yc})$	$\xi_{yc} = 0.844$

Flexural buckling check

		$N_{Ed} = 17 \cdot kN$	$M_{y,Ed} = 3.4 \cdot kNm$
[1] 5.4.4	$U_y := \left(\frac{N_{Ed}}{\chi_{\phi} \cdot \varphi \cdot N_{Rd}} \right)^{\xi_{yc}} + \frac{M_{y,Ed}}{\omega_o \cdot M_{y,Rd}}$		$U_y = 0.73$

Lateral-torsional buckling

[1] Annex J Figure J.2	Warping constant:	$I_w := \frac{(h-t_f)^2 \cdot I_z}{4}$	$I_w = 2.324 \cdot 10^8 \text{ mm}^6$
	Torsional constant:	$I_t := \frac{2 \cdot b \cdot t_f^3 + h \cdot t_w^3}{3}$	$I_t = 7.958 \cdot 10^3 \text{ mm}^4$
	Lateral buckling length	$L := l_{beam} - 2 \cdot b_b$	$W_y := W_{el}$ $W_y = 2.836 \cdot 10^4 \text{ mm}^3$

Constant bending moment, *warping* prevented at the ends

[1] H.1.2(2)	C_1, k and k_w constants	$C_1 := 1$	$k := 1$	$k_w := 0.5$
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[1] H.1.3(2)	$M_{cr} := \frac{C_1 \cdot \pi^2 \cdot E \cdot I_z}{(k \cdot L)^2} \cdot \sqrt{\left(\frac{k}{k_w}\right)^2 \cdot \frac{I_w}{I_z} + \frac{(k \cdot L)^2 \cdot G \cdot I_t}{\pi^2 \cdot E \cdot I_z}}$	$L = 921 \text{ mm}$	$M_{cr} = 9.026 \cdot kNm$
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[1] 5.6.6.3(3)	$\lambda_{LT} = \sqrt{\frac{\alpha_y \cdot W_y \cdot f_o}{M_{cr}}}$	$\lambda_{LT} = 0.988$
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[1] 5.6.6.3(2)	$\alpha_{LT} = \text{if}(\text{class}_z > 2, 0.2, 0.1)$	$\alpha_{LT} = 0.2$
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	$\lambda_{OLT} = \text{if}(\text{class}_z > 2, 0.4, 0.6)$	$\lambda_{OLT} = 0.4$
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[1] 5.6.6.3(1)	$\phi_{LT} = 0.5 \cdot \left[1 + \alpha_{LT} (\lambda_{LT} - \lambda_{OLT}) + \lambda_{LT}^2 \right]$	$\phi_{LT} = 1.047$
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	$\chi_{LT} = \frac{1}{\phi_{LT} + \sqrt{\phi_{LT}^2 - \lambda_{LT}^2}}$	$\chi_{LT} = 0.718$
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Check three sections	$i := 1..3$	$x_{s_1} := 0 \text{ mm}$	$x_{s_2} := x_{sw}$	$x_{s_3} := \frac{l_{beam}}{2} + r_b$
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$l_{zc} := l_{beam} + 2 \cdot r_b$	$\frac{x_s}{l_{zc}} = (0 \quad 0.135 \quad 0.5)$
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HAZ reduction factors	$(\omega_0 = 1 \text{ except at } x_{s_2})$
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[1] (5.51)	$\omega_i = \left(\rho_{haz} \cdot \frac{f_u}{\gamma} \cdot \frac{M1}{M2} \cdot \frac{\gamma}{f_o} \right)$	$\omega_i = \text{if}(\omega_i > 1, 1, \omega_i)$
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Weld at section $i = 2$	$\omega_i = \text{if}(i=2, \omega_i, 1)$	$\omega_0 = (1 \quad 0.717 \quad 1)$
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[1] (5.49)
or (5.52)

$$\omega_{\bar{x}} = \frac{\omega_0}{\chi_z + (1 - \chi_z) \cdot \sin\left(\frac{\pi \cdot x_s}{l_{zc}}\right)}$$

$\omega_x^T = (6.76 \quad 1.44 \quad 1)$

[1] (5.50)
or (5.53)

$$\omega_{xLT} = \frac{\omega_0}{\chi_{LT} + (1 - \chi_{LT}) \cdot \sin\left(\frac{\pi \cdot x_s}{l_{zc}}\right)}$$

$\omega_{xLT}^T = (1.39 \quad 0.86 \quad 1)$

[1] (5.42a)

$$\eta_{\bar{\theta}} = \alpha_z^2 \cdot \alpha_y^2 \quad \eta_{\bar{\theta}} := \text{if}(\eta_0 < 1, 1, \text{if}(\eta_0 > 2, 2, \eta_0)) \quad \eta_{\bar{\theta}} = 1$$

[1] (5.42b)

$$\gamma_{\bar{\theta}} = \alpha_z^2 \quad \gamma_{\bar{\theta}} := \text{if}(\gamma_0 < 1, 1, \text{if}(\gamma_0 > 2, 2, \gamma_0)) \quad \gamma_{\bar{\theta}} = 1$$

[1] (5.42c)

$$\xi_{\bar{\theta}} = \alpha_y^2 \quad \xi_{\bar{\theta}} := \text{if}(\xi_0 < 1, 1, \xi_0) \quad \xi_{\bar{\theta}} = 1$$

[1] 5.9.4.3

$$\eta_{\bar{c}} = \eta_0 \cdot \chi_z \quad \eta_{\bar{c}} := \text{if}(\eta_c < 0.8, 0.8, \eta_c) \quad \chi_{\bar{y}} = 0.844 \quad \eta_{\bar{c}} = 0.8$$

$$\gamma_{\bar{c}} = \gamma_0 \quad \chi_z = 0.148 \quad \gamma_{\bar{c}} = 1$$

$$\xi_{\bar{c}} = \xi_0 \cdot \chi_z \quad \xi_{\bar{c}} := \text{if}(\xi_{zc} < 0.8, 0.8, \xi_{zc}) \quad \xi_{\bar{c}} = 0.8$$

Lateral-torsional buckling of beam-column

$$M_{z,Ed} := 0 \cdot kNm$$

[1] (5.43)

$$U_{LT} := \left[\left(\frac{N_{Ed}}{\chi_z \cdot \omega_x \cdot N_{Rd}} \right)^{\eta_c} + \left(\frac{M_{y,Ed}}{\chi_{LT} \cdot \omega_{xLT} \cdot M_{y,Rd}} \right)^{\gamma_c} + \left(\frac{M_{z,Ed}}{\omega_0 \cdot M_{z,Rd}} \right)^{\xi_{zc}} \right] \quad U_{LT}^T = (0.535 \quad 1.071 \quad 1.104)$$

Max utilisation, lateral-torsional buckling $U_{z,max} := \max(U_{LT})$

$$U_{z,max} = 1.104$$

Max utilisation, flexural buckling

$$U_y = 0.73$$

Comment: $U_{z,max} = 1.104$ mean that the test failure load exceeded the characteristic strength with 10.4%